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# ON FINITE ELEMENT APPROXIMATION OF INCOMPRESSIBLE FLUID FLOW IN COMPUTATIONAL DOMAIN WITH VIBRATING WALLS: MATHEMATICAL MODELS FOR TREATMENT OF CHANNEL CLOSING \*

## PETR SVÁČEK $^{\dagger}$

**Abstract.** In this paper a simplified mathematical model of a voice production problem is considered. Here we focus on modelling of the glottis closure, which is an important part of phonation process. A simplified vocal fold model describing the vocal fold vibrations with two degrees of freedom is considered and coupled with a simplified model of the fluid flow described by the incompressible Navier-Stokes equations. The vocal fold vibrations cause a deformation of the fluid computational domain which is treated with the aid the Arbitrary Lagrangian-Eulerian method. The vibrations can possible lead to an appearance of the vocal folds contact. This situation is treated with the aid of a combination of inlet boundary conditions, a fictitious porous media approach and the Hertz impact forces. Numerical method is based on the stabilized finite element method. Numerical results are presented.

Key words. aeroelasticity, finite element method, contact problem

AMS subject classifications. 65N30, 76D05, 76Q05, 74M20

1. Introduction. Human phonation process is a complex phenomena consisting of the air flow, structural vibrations, their mutual interactions and periodical appearing of the vocal folds contacts, see [6]. Except this also the acoustics phenomena is important. One important aspect is that the phonation is in fact an aeroelastic instability. This aeroelastic instability causes vocal folds oscillations with large amplitudes leading to their mutual contact. This periodical contact leads to the closure of the vocal tract at glottal part. Thus a realistic mathematical model should consist of the fluid-structure interaction description as well as a mathematical model of the (periodical) contact of the vocal folds. As such a treatment is extremely difficult, usually a simplified models are employed, see e.g. the simplified two degrees of freedom model of the vocal folds of [3] or the aeroelastic model in [2].

This paper presents a complex mathematical model consisting of the fluid flow problem, the structural description by motion equations, coupling conditions and a treatment of the contact problem. For the fluid flow the model of incompressible Navier-Stokes equations is used written in the arbitrary Lagrangian-Eulerian(ALE) form. The acoustic modelling is omitted in this paper as the influence of acoustic forces on the fluid flow or on the structure motion is negligible in the voice production process.

Further, the numerical approximation based on the finite element (FE) method is described, where the attention is paid step by step to the time discretization, weak formulation, stabilization of the FE method, linearization and to the solution of the linearized problem. The described approach is applied for solution of a benchmark problem.

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<sup>&</sup>lt;sup>†</sup>Department of Technical Mathematics, Faculty of Mechanical Engineering, Czech Technical University in Prague, Karlovo nám. 13, 121 35 Praha 2, Czech Republic

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### 2. Mathematical model.

**2.1. Flow problem.** The air flow in terms of the flow velocity  $\boldsymbol{u} = (u_1, u_2)$  and the kinematic pressure p is modelled by the system of the Navier-Stokes equations (cf. [1]) written in the ALE form (cf. [4]), i.e. in the computational domain  $\Omega_t$  it is governed by

(2.1) 
$$\frac{D^{\mathcal{A}}\boldsymbol{u}}{Dt} + ((\boldsymbol{u} - \boldsymbol{w}_D) \cdot \nabla)\boldsymbol{u} = \operatorname{div} \boldsymbol{\tau}^f$$
$$\nabla \cdot \boldsymbol{u} = 0,$$

where  $\boldsymbol{\tau}^{f} = (\tau_{ij}^{f})$  is the fluid stress tensor given by  $\boldsymbol{\tau}^{f} = -p\mathbb{I} + 2\nu \boldsymbol{D}$ ,  $\boldsymbol{D}$  is the symmetric gradient tensor  $\boldsymbol{D}(\boldsymbol{u}) = \frac{1}{2}(\nabla \boldsymbol{u} + \nabla^{T}\boldsymbol{u})$  with components  $d_{ij} = \frac{1}{2}(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i})$  and  $\nu > 0$  is the constant kinematic fluid viscosity. In Equations (2.1)  $\boldsymbol{w}_{D}$  denotes the domain velocity and  $\frac{D^{A}\boldsymbol{u}}{Dt}$  is the ALE derivative, i.e. the derivative with respect to the reference configuration  $\Omega_{ref}$ , cf. [4].

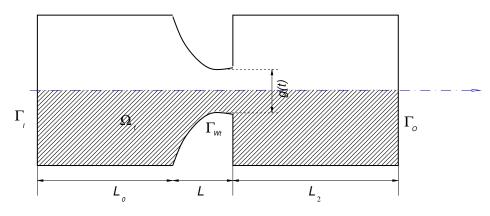


FIG. 2.1. The computational domain  $\Omega_t$  with specification of the boundary parts.

In order to solve system (2.1) an initial and mixed boundary conditions are prescribed at the boundary  $\partial \Omega_t$  of the computational domain. To this end it is assumed  $\partial \Omega_t$  is at any time  $t \in [0, T]$  formed by mutually disjoint parts  $\partial \Omega_t = \Gamma_I \cup \Gamma_S \cup \Gamma_O \cup \Gamma_t$ . Here,  $\Gamma_I$  denotes the inlet part of the boundary,  $\Gamma_O$  is the outlet part of the boundary,  $\Gamma_S$  denotes the axis of symmetry (in this paper it is part of *x*-axis y = 0 with the unit outward normal  $\mathbf{n} = (0, 1)$ ) and  $\Gamma_t = \Gamma_{Wt} \cup \Gamma_{Wf}$  denotes either the fixed ( $\Gamma_{Wf}$ ) or the deformable ( $\Gamma_{Wt}$ ) walls. The following boundary conditions are prescribed

(2.2)  
a) 
$$\boldsymbol{u} = \boldsymbol{w}_D$$
 on  $\Gamma_{Wt}$ ,  
b)  $u_2 = 0, -\tau_{12} = 0$  on  $\Gamma_S$ ,  
c)  $\frac{1}{2}(\boldsymbol{u} \cdot \boldsymbol{n})^- \boldsymbol{u} - \boldsymbol{n} \cdot \boldsymbol{\tau} = \frac{1}{\varepsilon}(\boldsymbol{u} - \boldsymbol{u}_I)$  on  $\Gamma_I$ ,  
d)  $\frac{1}{2}(\boldsymbol{u} \cdot \boldsymbol{n})^- \boldsymbol{u} - \boldsymbol{n} \cdot \boldsymbol{\tau} = p_{ref}\boldsymbol{n}$  on  $\Gamma_O$ ,

where  $\boldsymbol{n}$  denotes the unit outward normal vector to  $\partial\Omega_t$ ,  $\boldsymbol{u}_I$  is a prescribed inlet velocity,  $p_{ref}$  is a reference pressure value ( $p_{ref} = 0$  in what follows),  $\varepsilon > 0$  is a penalization parameter and  $\alpha^-$  denotes the negative part of a real number  $\alpha$ . Here, the boundary condition (2.2c) weakly imposes the Dirichlet boundary condition  $\boldsymbol{u} = \boldsymbol{u}_I$ with the aid of a penalization parameter  $\varepsilon$ .

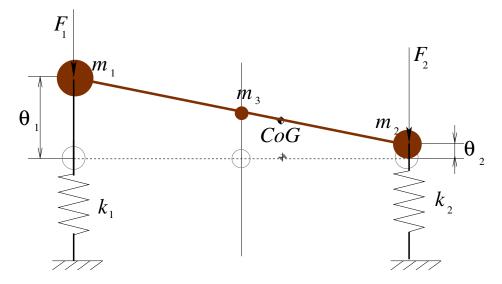


FIG. 2.2. Two degrees of freedom model (with masses  $m_1$ ,  $m_2$ ,  $m_3$ ) in displaced position (displacements  $\theta_1$  and  $\theta_2$ ) The acting aerodynamic forces  $F_1$  and  $F_2$  are shown.

**2.2. Structure model.** A simplified structural model of the vocal fold is used modelled as a rigid body consisting of three masses  $(m_1, m_2 \text{ and } m_3)$ , see Fig. 2.1. The vocal fold motion is governed by the displacements  $\theta_1(t)$  and  $\theta_2(t)$  of the two masses  $m_1$  and  $m_2$ . The equation of motion (see [2] for details) reads

(2.3) 
$$\mathbb{M}\ddot{\theta} + \mathbb{B}\dot{\theta} + \mathbb{K}\theta = -F.$$

where  $\boldsymbol{\theta} = (\theta_1, \theta_2)^T$ ,  $\mathbb{M}$  is the mass matrix of the system,  $\mathbb{K} = \text{diag}(k_1, k_2)$  is the diagonal stiffness matrix of the system characterized by spring constants  $k_1, k_2$ , and  $\mathbb{B} = \varepsilon_1 \mathbb{M} + \varepsilon_2 \mathbb{K}$  is the matrix of the proportional structural damping,  $\varepsilon_1$ ,  $\varepsilon_2$  are the constants of the proportional damping. The mass matrix is given by

(2.4) 
$$\mathbb{M} = \begin{pmatrix} m_1 + \frac{m_3}{4} & \frac{m_3}{4} \\ \frac{m_3}{4} & m_2 + \frac{m_3}{4} \end{pmatrix}.$$

The vector  $\mathbf{F} = \mathbf{F}_{imp} + \mathbf{F}_{aero}$  consists of the aerodynamical forces  $\mathbf{F}_{aero} = (F_1, F_2)^T$ and the Herz impact forces  $\mathbf{F}_{imp}$  due to the possible impact of vocal folds.

**2.3.** Coupling conditions. The aerodynamical forces  $F_1, F_2$  are evaluated with the aid of the aerodynamical lift force L(t) and aerodynamical torsional moment M(t) acting on the surface of the structure  $\Gamma_{Wt}$ . The aerodynamical lift force and the aerodynamical torsional moment are evaluated with the aid of the mean (kinematic) pressure p and the mean flow velocity  $\boldsymbol{u} = (u_1, u_2)$  as the integrals over the surface of the airfoil

(2.5) 
$$L = -l \int_{\Gamma_{Wt}} \rho \tau_{2j} n_j \, dS, \quad M = l \int_{\Gamma_{Wt}} \rho \tau_{ij} n_j r_i^{\text{ort}} \, dS,$$

where l denotes the depth of the profile section, and the vector  $r^{\text{ort}}$  has components  $r_1^{\text{ort}} = -(x_2 - x_2^{\text{EA}}), r_2^{\text{ort}} = x_1 - x_1^{\text{EA}}$  with  $(x_1^{\text{EA}}, x_2^{\text{EA}})$  being the position of the structure elastic axis.

The displacement of any point  $\boldsymbol{\xi} = (\xi_1, \xi_2) \in \Gamma_{Wt}$  is determined in terms of  $\theta_1, \theta_2$ as  $\mathcal{A}_t(\boldsymbol{\xi}) = (\xi_1, y)$ , where

(2.6) 
$$y = \xi_2 + \frac{\theta_1 + \theta_2}{2} + (\xi_2 - \frac{L_{ref}}{2})(\theta_2 - \theta_1)$$

This displacement is used as boundary condition for the sought ALE mapping  $\mathcal{A}_t$ , which maps the reference domain  $\Omega_0^{ref}$  onto the computational domain  $\Omega_t$ . Consequently the domain velocity at the surface  $\Gamma_{Wt}$  is determined by  $\theta_1, \theta_2$ , whereas the domain velocity in the interior of the domain  $\Omega_t$  needs to be evaluated as the time derivative of the ALE mapping  $\mathcal{A}_t$ .

2.4. Treatment of the contact. The use of the simplified structural model in combination with the symmetry assumption allows to calculate the gap g(t) between the vocal folds in terms of the initial gap g(0) and the values of displacements  $\theta_1(t)$  and  $\theta_2(t)$ , see Figure 2.1. Let us emphasize that the solution of the ordinary differential equations (2.3) formally allows this gap become zero or even negative - this situation corresponds to the impact of the vocal folds. On the other hand the displacement of the part  $\Gamma_{Wt}$  based on the displacements  $\theta_1(t)$  and  $\theta_2(t)$  (as part of the computational domain) is both geometrically not possible as well as physically incorrect (as during the impact the surface of the vocal fold is deformed at the contact area) for the case of g(t) being negative.

Moreover for the computational purposes (to avoid mesh distortion) it is also difficult to treat values of g(t) being still positive but close to zero. Consequently the deformation of  $\Gamma_{Wt}$  is treated with the aid of the formula (2.6) for the gap  $g(t) \ge g_{min} > 0$  (here  $g_{min}$  is usually specified as a small fraction of the initial gap). Two modifications of this formula are considered in the case when g(t) becomes lower then  $g_{min}$ .

The first approach is a simple vertical shift of the (rigid) vocal fold which keeps the actual (fictitious) gap g(t) equal to  $g_{min}$ . This means that the equation (2.6) is replaced by

(2.7) 
$$y = \xi_2 + \frac{\theta_1 + \theta_2}{2} - (g_{min} - g(t))^+ + (\xi_2 - \frac{L_{ref}}{2})(\theta_2 - \theta_1).$$

where  $(g_{min} - g(t))^+$  denotes the positive part of the number  $g_{min} - g(t)$ , i.e. it is either zero if  $g(t) \ge g_{min}$  or it shifts the whole surface to keep the the gap equal  $g_{min}$ otherwise.

The second more realistic approach is based on a modification of the displacement of only the points in the contact zone. This is realized by modification of Equation (2.6) for mesh vertices which violate the condition of  $g(t) \ge g_{min}$ , i.e. Equation (2.6) is modified as

(2.8) 
$$y = \min\left(\tilde{y}, g_{min}\right)$$

where  $\tilde{y}$  is computed by the original formula (2.6)

(2.9) 
$$\tilde{y} = \xi_2 + \frac{\theta_1 + \theta_2}{2} + (\xi_2 - \frac{L_{ref}}{2})(\theta_2 - \theta_1).$$

In both cases the computational fluid domain  $\Omega_t$  is formally decomposed into two parts,  $\Omega_t = \Omega_t^P \cup \Omega_t^f$  where by  $\Omega_t^f$  the domain occupied by fluid is denoted and by  $\Omega_t^P$  the part of the computational domain which should be occupied by the vocal tract (with no fluid) is denoted. The flow through the domain  $\Omega_t^p$  is modelled as flow through a fictitious porous media, see Fig. 2.3, i.e. the Navier-Stokes equations are modified by an added Darcy term  $\sigma_P \boldsymbol{u}$  in  $\Omega_t^P$ 

(2.10) 
$$\frac{D^{\mathcal{A}}\boldsymbol{u}}{Dt} + ((\boldsymbol{u} - \boldsymbol{w}_D) \cdot \nabla)\boldsymbol{u} + \sigma_P \boldsymbol{u} = \operatorname{div} \boldsymbol{\tau}^f$$

In practical realization Equation (2.10) is solved in the whole domain  $\Omega_t$  with  $\sigma_P = 0$ in  $\Omega_t^f$  and with a suitable chosen constant  $\sigma_P > 0$  in  $\Omega_t^P$ .

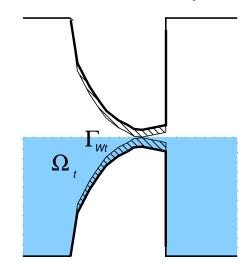


FIG. 2.3. The detail of the porous media flow domain  $\Omega_t^p$ .

3. Numerical approximation. The numerical discretization of the described coupled problem is realized by the stabilized finite element method applied for approximation of the fluid part, solution of the motion equations with the aid of 4th order Runge-Kutta method and coupling of both parts with the aid of a strongly coupled algorithm. For the purpose of the time discretization the time interval I is divided by an equidistant partition  $t_j = j\Delta t$  with a constant time step  $\Delta t > 0$ . The approximations of velocity and pressure at time instant  $t_j$  are denoted by  $\mathbf{u}^j \approx \mathbf{u}(\cdot, t_j)$  and  $p^j \approx p(\cdot, t_j)$  for  $j = 0, 1, \ldots$ . Similarly, by  $\mathbf{w}_D^j$  and  $\Omega^j$  approximations of the domain velocity  $\mathbf{w}_D(\cdot, t_j)$  and the computational domain  $\Omega_{t_j}$  at time instant  $t_j$  are denoted. In what follows we focus on numerical discretization at a (fixed) time step  $t_{n+1}$ . For the sake of simplicity the indices n + 1 are omitted in what follows, i.e. the following notation is used  $\mathbf{u} := \mathbf{u}^{n+1}$ ,  $p := p^{n+1}$ ,  $\mathbf{w}_D := \mathbf{w}_D^{n+1}$  and  $\Omega := \Omega^{n+1}$ .

**3.1. Flow problem.** In order to discretize equations (2.1) we start with approximation of the ALE time derivative at  $t = t_{n+1}$  by the second order backward difference formula

(3.1) 
$$\frac{D^{\mathcal{A}}\boldsymbol{u}}{Dt}|_{t_{n+1}} \approx \frac{3\boldsymbol{u}^{n+1} - 4\tilde{\boldsymbol{u}}^n + \tilde{\boldsymbol{u}}^{n-1}}{2\Delta t}$$

where at a given time instant  $t = t_{n+1}$  the velocity  $u^k$  defined on  $\Omega^k$  is transformed to the velocity field  $\tilde{u}^k$  defined on  $\Omega = \Omega^{n+1}$  by

(3.2) 
$$\tilde{\boldsymbol{u}}^{k}(x) = \boldsymbol{u}^{k}(\mathcal{A}_{t_{k}}(\mathcal{A}_{t_{n+1}}^{-1}(x))), \quad x \in \Omega.$$

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Further, the finite element discretization is based on the weak reformulation of time discretized equations (2.1). The function spaces for velocity and pressure are defined as  $\mathcal{V}$  and  $\mathcal{Q}$  given by

(3.3) 
$$\boldsymbol{\mathcal{V}} = \{ \boldsymbol{\varphi} \in \boldsymbol{H}^1(\Omega) : \ \boldsymbol{\varphi} \cdot \boldsymbol{n} = 0 \text{ at } \Gamma_S \}, \qquad \boldsymbol{\mathcal{Q}} = L_2(\Omega).$$

The space  $\mathcal{X}$  of test functions is subspace of  $\mathcal{V}$  specified as

(3.4) 
$$\boldsymbol{\mathcal{X}} = \{ \boldsymbol{\varphi} \in \boldsymbol{H}^1(\Omega) : \ \boldsymbol{\varphi} = 0 \text{ at } \Gamma_{W_t}, \ \boldsymbol{\varphi} \cdot \boldsymbol{n} = 0 \text{ at } \Gamma_S \}.$$

The weak form of Equations (2.1) is derived in the standard form: first, the ALE time derivative is replaced using the formula (3.1), next the first equation of (2.1) is multiplied by a test function  $\boldsymbol{z} \in \boldsymbol{\mathcal{X}}$ , integrated over  $\Omega$ , the Green's theorem for viscous terms and the pressure gradient is used and the boundary conditions (2.2) as well as the definition of spaces  $\boldsymbol{\mathcal{V}}, \boldsymbol{\mathcal{X}}$  are taken into an account. Similarly the second equation is multiplied by a test function  $q \in \boldsymbol{\mathcal{Q}}$ , integrated over  $\Omega$  and both equations are summed up together.

Thus we arrive to the weak form: Find  $U = (\boldsymbol{u}, p) := (\boldsymbol{u}^{n+1}, p^{n+1}) \in \boldsymbol{\mathcal{V}} \times \boldsymbol{\mathcal{Q}}$  such that  $\boldsymbol{u}$  satisfy the boundary condition (2.2a) and

holds for any  $V = (\mathbf{z}, q) \in \mathcal{X} \times \mathcal{Q}$ . The forms a and L are defined for any  $U = (\mathbf{u}, p) \in \mathcal{V} \times \mathcal{Q}$ ,  $\overline{U} = (\overline{\mathbf{u}}, \overline{p}) \in \mathcal{V} \times \mathcal{Q}$  and  $V = (\mathbf{z}, q) \in \mathcal{X} \times \mathcal{Q}$  as follows

$$a(\overline{U}; U, V) = \left( \left( \frac{3}{2\Delta t} + \sigma_P \right) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + c(\overline{U}; U, V) + \left( 2\nu \boldsymbol{D}(\boldsymbol{u}), \boldsymbol{D}(\boldsymbol{z}) \right)_{\Omega} + \frac{1}{\varepsilon} \left( \boldsymbol{u}, \boldsymbol{z} \right)_{\Gamma_I} + \left( \frac{1}{2} (\overline{\boldsymbol{u}} \cdot \boldsymbol{n})^+ \boldsymbol{u}, \boldsymbol{z} \right)_{\Gamma_I \cup \Gamma_O} + (\nabla \cdot \boldsymbol{u}, q)_{\Omega} - (\nabla \cdot \boldsymbol{z}, p)_{\Omega}$$
(3.6)

and

(3.7) 
$$L(V) = \left(\frac{4\tilde{\boldsymbol{u}}^n - \tilde{\boldsymbol{u}}^{n-1}}{2\Delta t}, \boldsymbol{z}\right)_{\Omega} + \frac{1}{\varepsilon} (\boldsymbol{u}_I, \boldsymbol{z})_{\Gamma_I} - \int_{\Gamma_O} p_{ref}(\boldsymbol{n} \cdot \boldsymbol{z}) dS,$$

where by the symbol  $(\cdot, \cdot)_{\mathcal{M}}$  the dot product in  $L_2(\mathcal{M})$  or  $L_2(\mathcal{M})$  is denoted. Further, the skew-symmetric trilinear form c represents the convection term (here we abbreviate  $\overline{\boldsymbol{w}} = \overline{\boldsymbol{u}} - \boldsymbol{w}_D^{n+1}$ )

$$(3.8) \ c(\overline{U};U,V) = \frac{1}{2} \left( (\overline{\boldsymbol{w}} \cdot \nabla) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} - \frac{1}{2} \left( (\overline{\boldsymbol{w}} \cdot \nabla) \boldsymbol{z}, \boldsymbol{u} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u}, \boldsymbol{z} \right)_{\Omega} + \frac{1}{2} \left( (\nabla \cdot \boldsymbol{w}_{D}^{n+1}) \boldsymbol{u} \right)_{\Omega} + \frac{1}{2} \left( ($$

In order to approximate problem (3.5) by finite element method, the spaces  $\mathcal{V}$ and  $\mathcal{X}$  are approximated using their FE subspaces  $\mathcal{V}_h$  and  $\mathcal{X}_h$ , respectively. These spaces are constructed over an admissible triangulation  $\mathcal{T}_\Delta$  of the domain  $\Omega$ . Similarly, the pressure space  $\mathcal{Q}$  is approximated by its FE subspace  $\mathcal{Q}_h$  constructed again over the same triangulation  $\mathcal{T}_\Delta$ . Here, the Taylor-Hood FEs are used, i.e. the spaces of continuous piecewise quadratic functions defined by

$$\mathcal{W}_h = \{ \varphi \in C(\overline{\Omega}) : \varphi \in P_2(K) \ \forall K \in \mathcal{T}_\Delta \}, \quad \mathcal{V}_h = \mathcal{W}_h \cap \mathcal{V} \quad \mathcal{X}_h = \mathcal{W}_h \cap \mathcal{X},$$

are used for velocities and the space of continuous piecewise linear functions

$$\mathcal{Q}_h = \{ \varphi \in C(\overline{\Omega}) : \varphi \in P_1(K) \ \forall K \in \mathcal{T}_\Delta \}.$$

are used for pressure approximations.

The FE approximations of  $\boldsymbol{u}_h \approx \boldsymbol{u}$  and  $p_h \approx p$  are then sought in the FE spaces  $\boldsymbol{\mathcal{V}}_h \times \mathcal{Q}_h$  constructed over an admissible triangulation  $\tau_h$  of the computational domain  $\Omega_t^f$ : Find an approximate solution  $U_h = (\boldsymbol{u}_h, p_h) \in \boldsymbol{\mathcal{V}}_h \times \mathcal{Q}_h$  such that Eq. (3.5) holds for any test function  $V_h = (\boldsymbol{z}_h, q_h) \in \boldsymbol{\mathcal{X}}_h \times \mathcal{Q}_h$ . Instead of formulation (3.5) the stabilized finite element approximations  $U_h = (\boldsymbol{u}_h, p_h)$  are sought in the space  $\boldsymbol{\mathcal{V}}_h \times \mathcal{Q}_h$  such that

(3.9) 
$$a(U_h; U_h, V_h) + \mathcal{P}(U_h, V_h) + \mathcal{S}(U_h; U_h, V_h) = L(V) + \mathcal{F}(U; V),$$

holds for any test function  $V_h = (\mathbf{z}_h, q_h) \in \mathcal{X}_h \times \mathcal{Q}_h$ . Here, the stabilization terms  $\mathcal{S}, \mathcal{F}$ and  $\mathcal{P}$  represents the SUPG/PSPG stabilization terms and the div-div stabilization terms, respectively. These terms are defined for any  $U_h = (\mathbf{u}_h, p_h) \in \mathcal{W}_h \times \mathcal{Q}_h$ ,  $\overline{U}_h = (\overline{\mathbf{u}}_h, \overline{p}_h) \in \mathcal{W}_h \times \mathcal{Q}_h$  and  $V_h = (\mathbf{z}_h, q_h) \in \mathcal{X}_h \times \mathcal{Q}_h$ 

$$\mathcal{S}(\overline{U}_{h}; U_{h}, V_{h}) = \sum_{K \in \mathcal{T}_{\Delta}} \left( \left( \frac{3}{2\Delta t} + \sigma_{P} \right) \boldsymbol{u}_{h} - \mu \Delta \boldsymbol{u}_{h} + \left( \overline{\boldsymbol{w}}_{h} \cdot \nabla \right) \boldsymbol{u}_{h} + \nabla p, \boldsymbol{\Psi}(V_{h}) \right)_{K}$$
  
(3.10)  $\mathcal{F}(\overline{U}_{h}; V_{h}) = \sum_{K \in \mathcal{T}_{\Delta}} \left( \frac{4\tilde{\boldsymbol{u}}_{h}^{n} - \tilde{\boldsymbol{u}}_{h}^{n-1}}{2\Delta t}, \boldsymbol{\Psi}(V_{h}) \right)_{K}$   
 $\mathcal{P}(U_{h}, V_{h}) = \sum_{K \in \mathcal{T}_{\Delta}} \tau_{K} \left( \nabla \cdot \boldsymbol{u}_{h}, \nabla \cdot \boldsymbol{z}_{h} \right)_{K},$ 

where  $\Psi(V_h) := \delta_K(\overline{w}_h \cdot \nabla) z_h + \delta_K \nabla q_h$ ,  $\overline{w}_h = \overline{u}_h - w_D^{n+1}$  and  $\delta_K$ ,  $\tau_K$  are suitably chosen stabilization parameters.

The problem (3.9) is nonlinear and requires an iterative solution. Here, the approach based on the Oseen linearization is used. Starting with an approximation  $U_h^0 \in \mathcal{V}_h \times \mathcal{Q}_h$  the linearized problems are solved for  $k = 0, 1, \ldots$ : Find  $U_h^{k+1} \in \mathcal{V}_h \times \mathcal{Q}_h$  such that

$$(3.11) \quad a(U_h^k; U_h^{k+1}, V_h) + \mathcal{P}(U_h^k, V_h) + \mathcal{S}(U_h^k; U_h^{k+1}, V_h) = L(V_h) + \mathcal{F}(U_h^k; V_h) =$$

holds for any  $V_h \in \mathcal{X}_h \times \mathcal{Q}_h$ . This process is repeated till  $||U_h^{k+1} - U_h^k|| < \varepsilon$  with a suitable chosen  $\varepsilon > 0$ .

**3.2. Structure model, ALE mapping and ALE derivative.** The motion equations (2.3) are formulated as a first order system and time discretized with the aid of 4th order Runge-Kutta method. This method is used to find the approximations  $\theta_1^n \approx \theta_1(t_n), \theta_2^n \approx \theta_2(t_n), \dot{\theta_1}^n \approx \dot{\theta_1}(t_n), \dot{\theta_2}^n \approx \dot{\theta_2}(t_n)$  starting from an initial condition. Here, the zero initial condition is used  $\theta_1^0 = \theta_2^0 = \dot{\theta_1}^0 = \dot{\theta_2}^0 = 0$  and the Runge-Kutta methods is applied on sub-intervals of the interval  $(t_n, t_{n+1})$ , where the aerodynamical forces  $F_1$  and  $F_2$  are interpolated in the interior of the interval from the values found at time instants  $t_n$  and  $t_{n+1}$ .

Based on the found displacements  $\theta_1^{n+1}$ ,  $\theta_2^{n+1}$  at the time instant  $t_{n+1}$  the displacement of the boundary  $\Gamma_{Wt_{n+1}}$  is determined by equation (2.6). The ALE mapping  $\mathcal{A}$ at time instant  $t_{n+1}$  is determined in terms of sought displacement of the reference domain with the Dirichlet boundary conditions specified by (2.6) at  $\Gamma_{Wref}$  and with the zero Dirichlet boundary condition otherwise. To this end the approach based on solution of a fictitious elastic problem is used similarly as in [7]. This approach however is strongly modified in order to keep the quality of the mesh satisfactory even

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in the case of almost enclosed channel. This is why, instead of solving of a linear elasticity problem, the geometrical non-linearities are taken into an account.

Once the ALE mapping at  $\mathcal{A}_{t_{n+1}}$  is determined, the ALE domain velocity  $\boldsymbol{w}_D^{n+1}$  is computed with the aid of the second order backward difference formula, i.e. for  $x \in \Omega^{n+1}$  with a reference  $\xi \in \Omega_{ref}$ , we have

(3.12) 
$$\boldsymbol{w}_D^{n+1}(x) \approx \frac{3x - 4x^n + x^{n+1}}{2\tau}$$

where  $x = \mathcal{A}_{t_{n+1}}(\xi), x^n = \mathcal{A}_{t_n}(\xi), x^{n-1} = \mathcal{A}_{t_{n-1}}(\xi).$ 

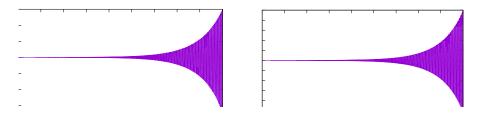


FIG. 4.1. The aeroelastic responses  $\theta_1(t)$  (left) and  $\theta_2(t)$  (right) of the structure for flow velocity  $U_{\infty} = 0.65$  m/s. The phonation onset phase is shown.

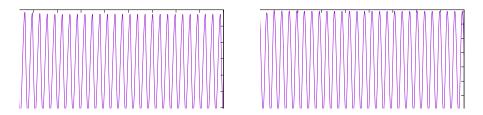


FIG. 4.2. The aeroelastic response of the structure for flow velocity  $U_{\infty} = 0.65$  m/s. The phonation phase with periodical closing of the gap between vocal folds and the Hertz impact forces involved.

4. Numerical results. In this section benchmark the problem from [5] is numerically approximated by the proposed method using both approaches for treatment of the channel closing. For the inflow velocity  $U_{\infty} = 0.65m/s$  the aeroelastic instability occurred are shown in terms of the displacements  $\theta_1$  and  $\theta_2$  in Figure 4.1. In the context of the voice production this corresponds to the phonation onset. With further continuation the amplitude vibrations increases, but the limitation by the gap guarantees these vibrations to stay limited, and leads to a limit cycle of oscillations as shown in Figure 4.2. Here, the vibrating vocal folds starts influenced by their mutual contact. Figure 4.3 shows the vibration onset the contact problem is unimportant (Figure 4.3, left), whereas during the phonation the gap between two vocal folds periodically becomes zero (Figure 4.3, right). This behaviour well corresponds to the results of simplified models. For a higher inflow velocity  $U_{\infty} = 0.7m/s$  similar behaviour was observed with much faster appearance of the phonation, see Fig. 4.4 and Fig. 4.5. Fig-

ures 4.6-4.7 then compares the two suggested approaches of geometrical gap closing in terms of flow velocity in the glottis region.

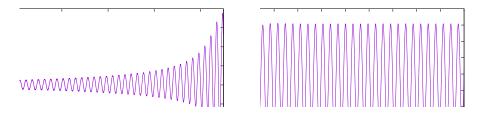


FIG. 4.3. The aeroelastic response of the structure for flow velocity  $U_{\infty} = 0.65$  m/s in terms of the gap g(t).

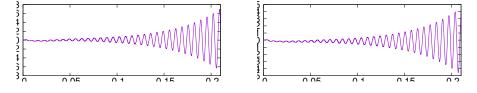


FIG. 4.4. The aeroelastic responses  $\theta_1(t)$  (left) and  $\theta_2(t)$  (right) of the structure for flow velocity  $U_{\infty} = 0.70$  m/s - phonation onset

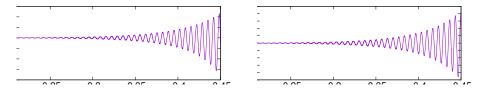


FIG. 4.5. The aeroelastic responses  $\theta_1(t)$  (left) and  $\theta_2(t)$  (right) of the structure for flow velocity  $U_{\infty} = 0.70$  m/s. The phonation onset phase is shown with much faster growth of the amplitudes.

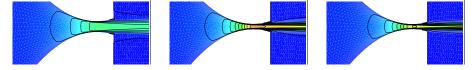


FIG. 4.6. The flow velocity magnitude during the opening and closing phase for the inlet flow velocity  $U_{\infty} = 0.65$  m/s, simple rigid body approach of closing applied

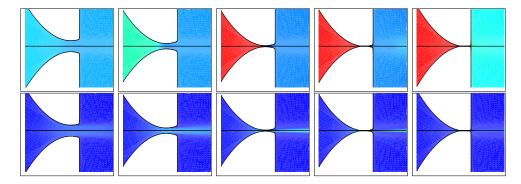


FIG. 4.7. The flow velocity magnitude during the opening and closing phase for the inlet flow velocity  $U_{\infty} = 0.65 \text{ m/s}$  - second approach

5. Conclusion. This paper presents the detailed description of the numerical approximation of the problem of fluid-structure interaction problem used in models of human phonation. Main attention is paid to modelling of the contact of vibrating vocal folds. Here two strategies are suggested and numerical results are shown.

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